

Title: COMPOSITE LINER DESIGN TO MAXIMIZE THE SHOCK PRESSURE BEYOND MEGABARS

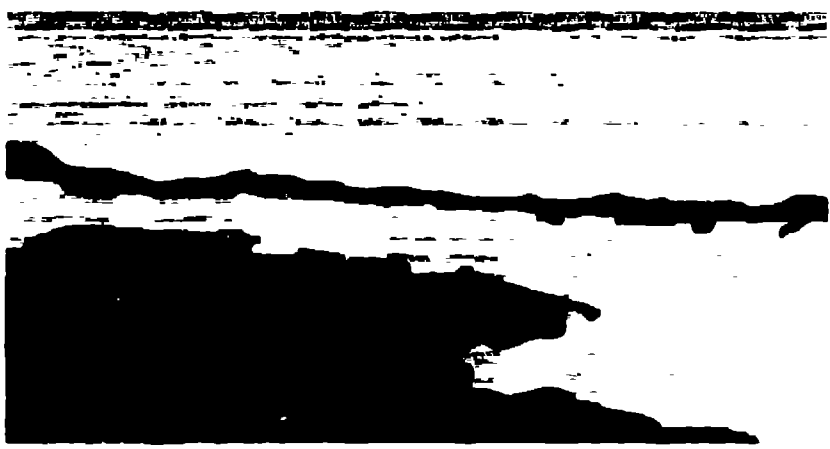
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## COMPOSITE LINER DESIGN TO MAXIMIZE THE SHOCK PRESSURE BEYOND MEGABARS

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*Among the solid liners made of a single material which are imploded onto a target under the same driving condition, the aluminum liner produces the highest shock pressure. We propose the composite liner design which can increase the shock pressure several times over the the best performance obtainable from an aluminum liner. We have also developed a general formulation to optimize the composite liner design for any driving current, and derived a set of very useful scaling relations. Finally, we present some 1-D simulations of the optimal composite liners to be fielded at Pegasus and Procyon in the upcoming megabar experiments.*

### I. Introduction

Using pulsed power to implode the liner onto a target is a convenient way to produce high shock pressure. Two years ago a solid aluminum liner which could produce shock pressures in the hundreds of kbar regime was designed [1] and tested [2] at the LANL Pegasus facility. This liner design has since been used successfully for a variety of application experiments. Recently, there have been considerable experimental interests to produce shock pressures in the megabar regime and quite importantly, to determine what is the practical pressure limit attainable for a given pulsed power. It will be made clear later that, among the liners made of a single material (for convenience called pure liner in this paper), the aluminum one is generally the best to produce the highest shock pressure. Here we scale up the solid aluminum liner design to the maximum driving current at Pegasus, the highest pressure we can get is just around 2 Mbar.

In this paper we propose a composite liner design which can increase the shock pressure several times over the best performance of the aluminum liner. The composite liner is made of aluminum on the inside and platinum on the outside. Other practical considerations through a systematic study of the shock pressure behavior indicate that the Hugoniot and a general analysis of the inside loading history is a suitable method. We have already developed a useful formulation to optimize the composite liner design for any given driving current. Using the aluminum liner as a equivalent with the driving current scaled from the best solid liner case, we have derived a set of useful scaling relations for the optimal liner

parameters and performance. These relations provide us a quick benchmark estimate on the maximum pressure attainable from any scalable driving current. For example, we can scale our results obtained for Pegasus to the Atlas parameter regime, since the driving currents for both are approximately sinusoidal.

In the next section we will present the liner implosion equation to lay the groundwork for liner design. We next discuss the general behavior of the shock pressure through the Hugoniot in Sect. III. We establish a shortcut for searching the best loading materials which maximize the shock pressure. In Sect. IV we give a detailed discussion on the Joule-heating, which may significantly limit on the liner velocity. The physical considerations leading to the composite liner and general procedure to optimize the liner parameters are given in Sect. V, followed by the derivation of the scaling relations in Sect. VI. Finally, in Sect. VII we discuss the optimized composite liners to be fielded in the upcoming megabar liner experiments at both Pegasus and Procyon, and we present some 1-D simulation results.

### II. Implosion Equation for Thin Liner

The liner implosion problem is rather complicated for the following two reasons. First, the driving current is not independent of the liner radius. This means that the equation of motion must be coupled to the circuit equation (XIV) in a self-consistent treatment. Second, the liner thickness is time dependent due to the radial expansion, which complicates the 1-D distribution. Since we are mainly interested in problems to optimize the liner design, we will neglect the effect of the time

Rather, we will take advantage of any good approximation which helps to simplify the implosion equation and render the scaling possible. The thin-liner approximation will be assumed in this paper, it is justified if the thickness of the liner is much smaller than the radius:

Next we note that the liner radius affects the driving current only through a logarithmic term in the inductance, so the effect is negligible until the liner radius  $r$  becomes much smaller than its initial value  $r_0$ . In the region where  $r \ll r_0$ , the duration is so short that the liner velocity is affected only slightly by the error in current. The above reasoning justifies that we can decouple the driving current from the liner motion. This excellent approximation not only simplifies greatly the implosion equation but also makes the scaling of the optimal liner parameters possible. Using the above approximations, the liner implosion equation is given by

$$r(t) = -\left(\frac{\mu_0}{4\pi}\right) \frac{I^2(t)}{mc(t)} \quad (1)$$

with the initial conditions  $r(0) = r_0$  and  $\dot{r}(0) = 0$ , where  $L$  is the length,  $I(t)$  the driving current,  $m$  the mass, and  $c(t)$  the mass of the liner.

A class of currents is said to be scalable to one another if we can represent them by a single function as  $I_p F(\omega t)$  using two parameters  $I_p$  and  $\omega$ . The current wave forms we usually see in many pulsed power are  $n_1$  approximately sinusoidal or like a step function, each type forms a scalable class. Later when we look for possible scaling relations of the optimal liner parameters and performance for scalable driving currents, it is useful to express the implosion equations for the whole class in terms of the scaled distance traveled by the liner,

$x \equiv (r_0 - r)/r_0$ , and scaled time  $\tau \equiv \omega t$ . The resulting implosion equation

$$x = \frac{\omega I_p^2(r)}{4\pi m c(\tau)} \quad (2)$$

has the an invariant set of initial conditions  $x(0) = 0, \dot{x}(0) = 0$ , where the dot stands for  $d/d\tau$  and

$$c = \left(\frac{r_0}{r}\right) \frac{I_p^2}{4\pi m c_0} \quad (3)$$

Let  $\tau_0$  be the scaled collision time, then the collision velocity is given by

$$v_c = r_0 \omega \dot{x}(\tau_0) \quad (4)$$

### III. Behavior of Shock Pressure Substituted from Hugoniot's

When the liner collides with the target at a velocity  $v_c$ , the shock pressure can be related to

the Hugoniot for the liner (labeled by  $l$ )

$$P_l(v) = \rho_l v(c_l + s_l v) \quad (5)$$

and target (labeled by  $t$ )

$$P_t(v_c - v) = \rho_t (v_c - v)(c_t + s_t(v_c - v)) \quad (6)$$

by eliminating the particle velocity  $v$ , where  $\rho$  is the density and  $c$  and  $s$  are material constants that relate the shock velocity to the particle velocity. From the above equations we see that higher collision velocity  $v_c$  and material densities will give rise to higher shock pressure, but the material with higher values in  $c$  and  $s$  also helps. While the above equations provide us a precise guideline to find the best liner and target materials that will achieve the highest collision shock under a given impinging condition, the process to examine all material pairs will be extremely time consuming. Fortunately we can take a shortcut by proving the following theorem: for any collision velocity let  $P_{AB}$  be the shock pressure generated from a collision between two materials A and B, its value is bounded in between  $P_{AA}$  and  $P_{BB}$ . We can prove this statement as follows. First, notice that the Hugoniot's are parabolic functions of  $v$ . In the physical region  $0 \leq v \leq v_c$ , the liner Hugoniot increases while the target Hugoniot decreases with increasing  $v$ . Second, for the collision between identical material, the two Hugoniot's always intersect at  $v = v_c/2$  due to their reflection symmetry at the point, so we have the exact relation

$$P_{AA} = \frac{1}{2}(P_{AA} + P_{BB}) \quad (7)$$

Without loss of generality, we assume  $P_{AA} > P_{BB}$  and solve for  $P_{AB}$  at the intersection of the liner Hugoniot A and target Hugoniot B. Now  $P_{AA}$  and  $P_{AB}$  lie on the liner Hugoniot A which increases with  $v$ , and  $P_{BB}$  and  $P_{AB}$  lie on the target Hugoniot B which decreases with  $v$ . It follows that  $P_{AB}$  can only occur in the region  $v_c/2 < v < v_c$  and therefore

$$P_{AA} > P_{AB} > P_{BB} \quad (8)$$

Using the above theorem, we can roughly set task sensibly in searching for the right material to maximize the shock pressure. Instead of searching for the maximum of all  $P_{AB}$ , we can just look for the maximum of  $P_{AA}$ , provided that the highest attainable  $v_c$  is independent of the liner material. We will show later that the last condition can be fulfilled for the composite liner.

From the material implosion equation (Eq. 1) and (4) for any  $v_c$ ,  $I_p$  is given as a function of the collision

material always maximizes the pressure for all values of  $v_c$ . But for multi-megabar pressures or higher, this is indeed the case. This follows from the fact that, for a wide variety of materials [4],  $c$  is around a few  $\text{mm}/\mu\text{s}$  and  $1.2 < s < 2$ . At high megabar pressures,  $v_c$  is large enough so that the term  $sv_c$  dominates over  $2c$  in Eq.(i) and consequently we have

$$P \approx \frac{1}{4} \rho s v_c^2. \quad (9)$$

This ensures that  $P$  is the maximum for the material with the highest value in  $\rho s$  at any  $v_c$ . In the same approximation, the shock pressure between two different materials behaves like

$$P_{AB} \approx \frac{v_c^2}{[(s_A \rho_A)^{-1/2} + (s_B \rho_B)^{-1/2}]^2} \quad (10)$$

#### IV. Joule Heating Limitation on Liner

The current passing through the liner has to diffuse into its interior from the outer surface, so calculating the resistive heating of the liner is quite complicated unless the diffusion time is faster than the implosion time. In general we do not expect the temperature distribution across the liner to be uniform, but rather to increase monotonically toward outside. To simplify the formulation, let us consider a pure liner and assume that the temperature is uniformly distributed. Since radiation loss is negligible, the time dependence of the liner temperature is given by the energy balance equation

$$R(t)I^2(t)dt = mc(T)dT, \quad (11)$$

where  $c$  is the specific heat of the liner material and  $R$  the resistance. In term of the resistivity  $\eta$  and density  $\rho$ , we can integrate the above as

$$\frac{\ell^2}{m^2} \int_0^t I^2(t)dt = \int_{T_0}^{T(t)} \frac{c(T)\rho(T)}{\eta(T)} dT, \quad (12)$$

where  $T_0$  is the initial temperature. Notice that the right hand side is only a state function of the liner material. The left hand side is proportional to the electrical action integral defined as

$$Q(T(t)) = \frac{1}{\lambda^2} \int_0^t I^2(t)dt, \quad (13)$$

where  $\lambda$  is the liner cross section. The electrical action for any conductor can be measured by passing a current through a thin sample wire. Setting a limit on the action by requiring

$T(t_c) = T_c$ , we constrain the liner mass to be a function of the collision time  $t_c$  as

$$\frac{\ell^2}{m^2} \int_0^{t_c} I^2(t)dt = \frac{Q(T_c)}{\rho^2}. \quad (14)$$

For pure liners, a reasonable limit on  $T_c$  is the melting point  $T_m$ , since the solid phase maintains a sharp shock front. The relation derived in Eq.(14) is still useful even when we deal with the realistic situations in which the temperature distribution is not uniform. In this case we should set the limit on the temperature of the inside liner surface, denoted by  $T(t)$ , which is the coolest at any time since the current has to diffuse radially inward. It is easy to see that we can still write

$$\frac{1}{m^2} \int_0^t I^2(t)dt = \beta(T(t), m), \quad (15)$$

except that  $\beta$  now has a weak dependence on  $m$ . Once we set  $T$  to a limit  $T_c$  at  $t = t_c$ ,  $\beta(T_c, m)$  can be determined by using the 1-D MHD code to compute the left hand side of Eq.(15). Later when we apply the above relation to optimize the liner mass, we only need to vary  $m$  in a narrow range around the optimal solution. We can therefore represent  $\beta(T_c, m)$  as a constant plus a small linear term in  $m$  and determine it by just two code simulations.

Among all metals, empirically aluminum has the highest value (only copper is a close second) in the ratio  $Q(T_m)/\rho$ , where  $Q(T_m)$  is the action to the melting point. In terms of  $Q(T_m)/\rho^2$ , the aluminum is ahead of other heavier metals even more by an extra density factor. Using Eq (14) the same can be said about the current integral on the left hand side. We therefore conclude that the aluminum liner can be driven with a longer  $t_c$ , before reaching the melting point, than any other pure liner (of higher density) having the same mass  $m$  and length  $\ell$ . But longer implosion time before melt implies higher attainable velocity since all these liners are governed by the same implosion equation. Using Eq (10), we see that this  $1/\rho$  advantage in attainable velocity for aluminum over materials of higher density is sufficient to ensure that the aluminum liner will also generate the highest shock pressure on any chosen target.

#### V. Composite Liner and Optimization

With the physical insights gained from our discussions on shock Hugonots and Joule heating, the composite liner seems to be an excellent idea to improve the attainable shock pressure substantially over the pure liners. Clearly we still want to use

aluminum on the outside for carrying most of the driving current to retain its highest attainable velocity. For the inner layer we look for a material with high value in  $\rho_a$  to enhance the shock pressure, subject to some other criteria discussed below.

We find that platinum is the best impacting material for the composite liner, not just for its high density but also for its high melting point and electrical resistivity. Based on these criteria, other materials such as tungsten are equally satisfactory, but the fact that platinum can be electroplated is a big plus for fabrication. The Joule heating in the platinum layer is reduced dramatically since the current has to diffuse in through the aluminum. The sheet resistance of the platinum layer is two orders of magnitude higher than that of the aluminum, owing to a much higher resistivity and smaller cross section. This factor also helps to reduce the Joule heating in Pt after the current is diffused in.

The high melting point is an extra advantage since we can now drive the Al layer beyond its melting point while still keeping the Pt layer solid. Consequently, the composite liner can take considerably more Joule heating than a pure aluminum one with the same mass, and thereby achieve a corresponding higher velocity. How much we can push this advantage depends on the ability of the solid Pt layer to withstand the magnetically-driven Rayleigh-Taylor instabilities in the molten Al layer. No definitive answer has been known so far from the 2-D MHD simulations. If fully, we will get some valuable clue from the upcoming megajoule liner experiment at Fegauke.

For the composite liner, clearly the Joule heating constraint should be applied to the aluminum layer. In applying Eq. (14), the factor  $m$  is replaced by the aluminum mass  $m_a$ , and  $Z$  is  $C$ , the melting temperature, not on its inner surface. The platinum mass  $m_p$  should be kept as low as practical so that it will not reduce the liner velocity significantly. In the following,  $v_{opt}$  is a specified velocity.

To optimize the liner design means to find the liner plane and radius which maximize the shock pressure at a given target radius  $r_0$ , which is usually determined by the experimental requirement or engineering limitation. For a  $v_{opt}$  as a function of  $r_0$ , we can find  $r_0$  and  $v_{opt}$  as a function of  $r_0$  and  $v_{opt}$  by using a given approximation. The approximation of  $v_{opt}$  that we used will be presented later, where, for a general current wave form, we optimize the liner parameters numerically as follows. Taking the liner

mass  $m$  as the free parameter  $r$ , we use Mathematica to solve Eq. (1) iteratively to find the correct initial radius  $r_0(m)$  such that the solution for  $r(t)$  satisfies  $r(t_c) = r_0$ , where  $t_0(m)$  is given by the Joule heating constraint Eq. (15). The optimal liner is then the one which maximizes the collision velocity  $v_2(m)$ . The result is then used in the 1-D MHD code to compute the more accurate liner motion and detailed shock pressure history. In spite of the thin-liner approximation and using a current-independent current in our formulation, the code simulations (with coupled circuit model) have demonstrated that we hardly need to refine the optimal liner parameters.

## VI. Scaling Relations for Optimal Liners

We now proceed to derive a set of very useful scaling relations for the optimal liner parameters and performance. While these relations are derived under some idealized scaling conditions, they nevertheless provide us a valuable benchmark to make a good estimate on the maximum pressure achievable by an unexplored pulsed power regime that is usually approximately scalable to a certain one.

It is important to realize that the liners optimized by the procedure as described in Sect. V do not scale in a simple way even though the driving currents are exactly scalable. For one thing, in realistic design, the target radius  $r_0$  is usually determined by experimental requirements so the ratio  $r_0/m$  will not stay the same from one driving condition to the other. Furthermore, the optimization requirement also complicates the scaling. Therefore, some idealized conditions are necessary for us to deduce a set of simplified but adequate scaling relations. In this end, we have to assume first that the implosion distance  $(r_0 - r_{in})$  scales like the optimal liner radius  $r_0$ . In terms of the scaled distance introduced in Eq. (2),

$r_0 - r_{in} = r_0/r_0$  stays constant. However, Eq. (2) is generally valid for any optimized liner design, the  $r_0/r_0$  is in fact not exactly constant.

Next we require that the scaled collision time defined by  $r_0/v_2$  must stay constant. This is equivalent to the condition that the kinetic energy for the liner at collision contains a constant fraction of the total driving energy. While we are unable to prove rigorously that this condition ensures the scaled liner parameters to be exactly optimal, physically it is far very reasonable. Moreover, its validity is proven indirectly by comparing the scaling results with the actual design conditions in actual experiments. We will

that the solutions for Eq (2) with different values of  $\alpha$  do not intersect except at  $\tau = 0$ , so there is only one solution which passes through  $x = r_2$  at  $\tau = \tau_c$  as required. Using Eq (3) the unique value of  $\alpha$  implies

$$m r_2^2 \propto I_p^2 \omega^{-2} \quad (16)$$

The Joule heating constraint given by Eq. (15) can be written as

$$\frac{I_p^2}{m^2 \omega} \int_0^{\tau_c} F^2(\tau) d\tau = \beta(T_1, t_0) \quad (17)$$

when we ignore the small amount of platinum mass. Neglecting the weak  $m$  dependence in  $\beta$  we get the scaling for the liner mass as

$$m \propto I_p \omega^{-3/2} \quad (18)$$

From Eqs (16) and (18) we obtain the scaling for the liner radius as

$$r_2 \propto I_p^{1/2} \omega^{-1/4} \quad (19)$$

Finally,  $\alpha(\tau_c)$  is constant and using Eq (4) we get

$$v_c \propto I_p^{1/2} \omega^{1/4} \quad (20)$$

for the collision velocity and

$$P \propto I_p \omega^{1/2} \quad (21)$$

for the shock pressure. As mentioned earlier, we have  $m r_2^2 \propto I_p^2$  to verify that the liner kinetic energy at collision is indeed a constant fraction of the total driving energy.

In applying these scaling relations, we can always if necessary, refine the result by taking the total expansion due to the imploding cylinder and liner deviation from axial scaling in either current or collision radius.

## VII. Megabar Liners for Pegasus and Procyon

Our composite liner design has been adapted to the upcoming megabar liner experiments at both Pegasus and Procyon. The experiments will be for the Pegasus megabar of 8 kg of aluminum and 1 kg of platinum, and it has an inner radius of 3 mm and a length of 2 cm. The liner design is based on the target radius  $r_2$  at 1 cm using the maximum driving current as shown in Fig. 1. Notice that the current is well represented by a sine curve up to the collision time of 880 ns. From the 1-D simulation we expect a peak shock about 8 Mbar on a platinum target. In Fig. 2 we display the liner design. The platinum layer is less than 1  $\mu$ m thick. In the plot we illustrate with the most

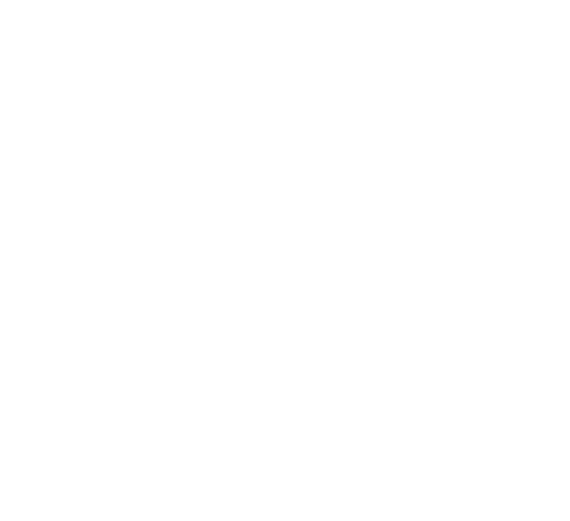


Figure 1. Implosion current for Pegasus megabar liner experiment.

radius of the aluminum layer as a single curve in the plot.

In Fig. 3 we show the velocity of the inner liner surface. The collision velocity is 8 mm/ $\mu$ s.

The temperature histories for the two liner layers are shown in Fig. 4. The dashed and solid (dotted and dashed) curves represent, respectively, the outer and inner surfaces of the platinum (aluminum) layer. We note that the inner aluminum surface begins to melt at 4  $\mu$ s and we draw the inner one about 7.5  $\mu$ s, but the platinum layer remains well below its melting point before collision.

Finally, in Fig. 5 we plot the shock pressure profiles against the zone number in four different cases. The platinum target covers the zones from 1 to 20, platinum in the liner from 20 to 12 and Al from 11 to 102. The solid curve is right after the collision at  $t = 880$  ns, followed by the dotted and dashed curves at 7 ns intervals and then the solid and long dashed curves at various intervals. The peak pressure is around 8 Mbar and stays slightly less than 8 ns. The peak shock duration is limited by the width of the Pt layer in the liner, which determines the length of the target. Iron wires coating the Pt-Al interface to reach the collision interface.

Procyon is an explosive driven pulsed power facility at LANL. The current waveform is approximately a sine function, with a sharp rise time about 1  $\mu$ s and peak current of 23 MA. Based on the driving current and the target system set at 1 cm, the optimum design parameters are listed in Table



Figure 2. Inset and outer liner radii versus time

of Al and 1 g of Pt, and has a length of 2 cm and an inner radius of 3.3 cm. According to our 1-D simulation, this liner will generate a peak shock about 20 Mbar on a Pt target.

## 1. Conclusions

We have proposed the aluminum-platinum composite liner design based on the physical insights obtained from our study of the behavior of the Hugoniot and electrical resistivity of various materials. The composite liner enables us to achieve a shock pressure several times over the best performance attainable if at the solid aluminum liner. This improvement in peak shock pressure results mostly due to the high density of the platinum layer, and to a lesser extent from the fact that we can keep the platinum layer in solid phase while driving the aluminum layer to past its melting point and thereby achieving an even higher collision velocity than the pure aluminum liner. As we push the design toward that of a fully liquid outer shell to see if we can just how well the solid platinum layer is able to withstand the Rayleigh-Taylor instability and shield the melted aluminum layer. Hopefully the upcoming neutron experiments at both LBNL and Princeton will shed some light on this question. We have developed a general formulation to optimize the composite liner design so that the shock pressure is maximized by simply trying out to change the densities and mass fractions of the materials used for the inner and outer layers of the optimized liner up until the

Figure 3. Velocity history of the shock liner surface

best performance.

## II. Reference

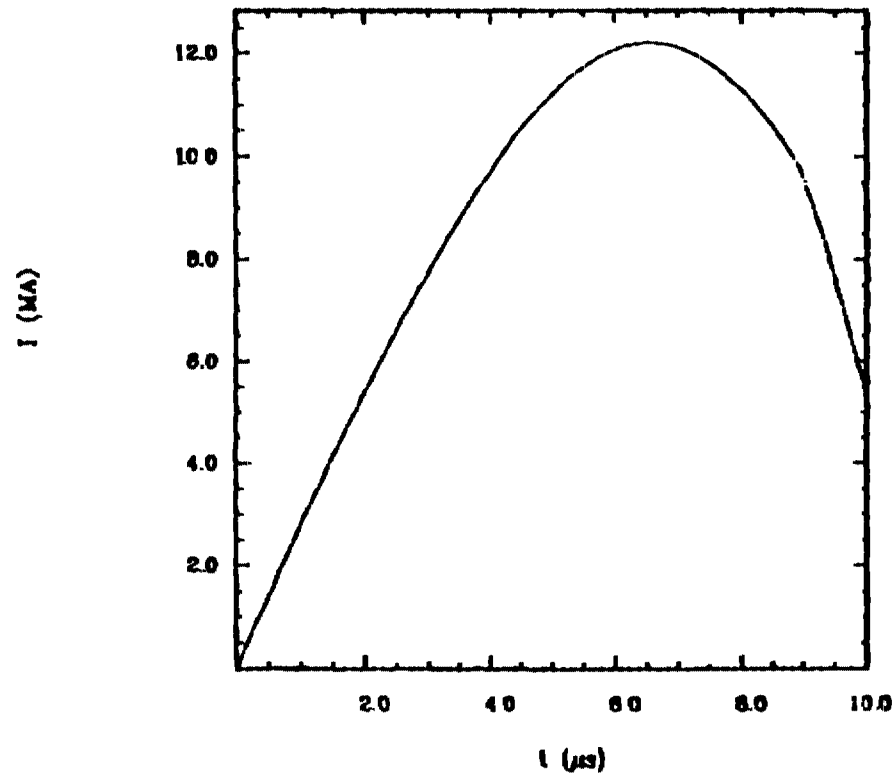
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Figure 4. Temperature histories of the outer (dotted and dashed) and inner (dot dash and solid) surfaces of the Al and Ti layers, respectively.

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Figure 5. Shock pressure profiles plotted versus time (number right after the collision) for (left) and at a later time (left) after 700 ns (right) and 1.00 ns (right) after impact.

Fig 1 H. Lee



R (cm)

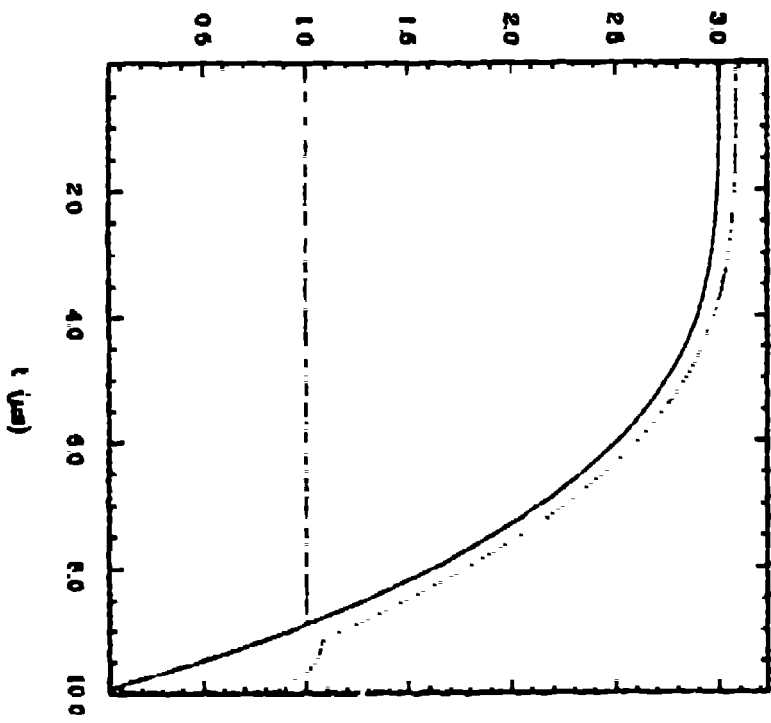


Fig 3

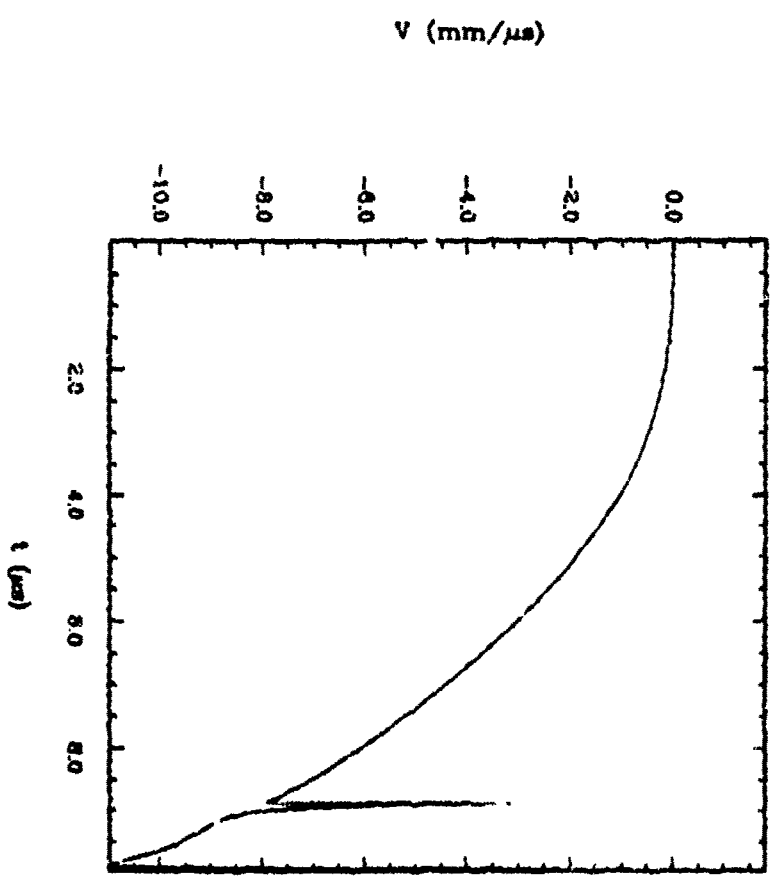


Fig 4

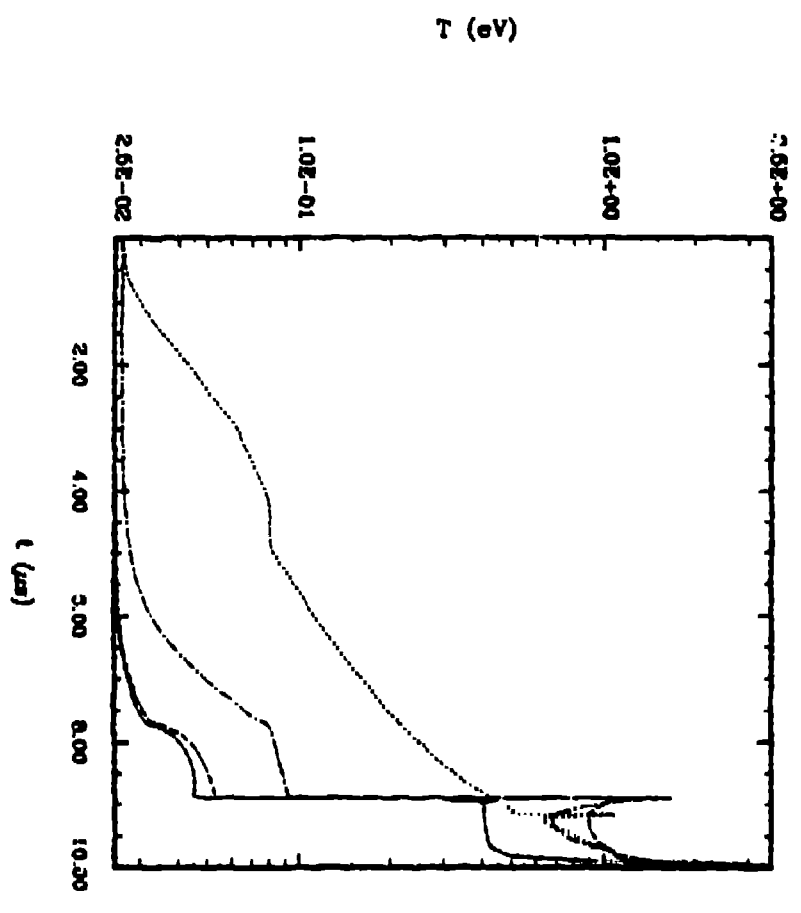


Fig 5

